

VIBRATION TYPE DRIVING APPARATUS

BACKGROUND OF THE INVENTION

Field of the Invention

5 This invention relates to a vibration type driving apparatus in which a vibration member of which the vibration is excited and a contacting member contacting with the vibration member move relative to each other.

Related Background Art

10 An already proposed vibration type driving apparatus is constructed as shown, for example, in Fig. 6 of the accompanying drawings. This vibration type driving apparatus has a ring-like vibration member 61 and a moving member (contacting member) 74.

15 The vibration member 61 is comprised of an elastic member 63, a group of piezoelectric elements 64 adhesively secured to the underside of the elastic member 63, and a friction member 66 adhesively secured to the upper surface (contacting surface) of the comb-tooth-like portion of the elastic member 63, and the
20 underside of the moving member 74 is brought into pressure contact with the upper surface (sliding surface) of the friction member 66 by pressing means, not shown.

25 In the vibration type driving apparatus thus constructed, when a periodic signal is applied to the group of piezoelectric elements 64, a travelling wave

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in which elliptical motions having a plurality of time phase differences are combined together is excited in the surface particles of the vibration member 61 (the elastic member 63 and the friction member 66), and the
5 moving member 74 is frictionally driven and rotated. Accordingly, the frictional contact state between the vibration member 61 and the moving member 74 is a factor which determines various kinds of performance such as the output torque, the number of revolutions,
10 the energy efficiency and the life of the driving apparatus.

So, heretofore, polishing work (see Japanese Patent Application Laid-Open No. 2-211074) or grinding work (see Japanese Patent Application Laid-Open No.
15 1-286783) has been done to make the flatness and surface roughness of the sliding surface of the friction member 66 fall within a predetermined range. There is also a case where the portion around the sliding surface of the friction member 66 is worked
20 into a predetermined shape by cutting work (see Japanese Patent Application Laid-Open No. 6-46580).

In the polishing work and the grinding work, however, the handling of the friction member 66 is inconvenient and requires a long working time and a
25 high cost. Also, in the polishing work and the grinding work, irregularity is liable to occur to the finished state of the sliding surface, and this is

considered to directly affect the irregularity of the performance of the vibration type driving apparatus.

Also, the cutting work requires many steps and moreover, the work must be done individually for each member, and in addition, to finish the cut shape between workpieces uniformly, it is necessary to strictly effect the custody of a parameter such as a working force, and this is inconvenient and takes a long working time.

Further, the friction member 66 popular in the vibration type driving apparatus, as shown in Fig. 7 of the accompanying drawings (a cross-sectional view taken along the line 7 - 7 of Fig. 6), is formed of a hard additive 84 and relatively soft binder resin 85 and therefore, when it is cut by a cutting tool, cutting resistance increases in the portion of the additive and decreases in the portion of binder resin. Therefore, the cut surface becomes an uneven surface in which the additive protrudes from the binder resin, and a suitable smooth surface is difficult to obtain.

As regards other material of the friction member, for example, in Japanese Patent Application Laid-Open No. 5-239442 and Japanese Patent Application Laid-Open No. 4-49872, a resin material containing carbon fiber is disclosed as a material being good in abrasion property and suitable as a frictional material for a vibration wave motor from the long durability life and

the stability of the coefficient of friction thereof.

However, when the evaluation of the performance of the vibration wave driving apparatus under various environments has been effected in the various uses of the vibration wave driving apparatus, it has been found that the abrasion resistance of a frictional material, i.e., the abrasion loss of the frictional material, is remarkably changed by the influence of a change particularly in humidity. Therefore, it has been found that the life of the vibration wave driving apparatus depends greatly on the durability life of the frictional material and is greatly changed under the influence of a change in the humidity particularly in the environment of use.

A frictional material for a vibration wave driving apparatus stable against the influence of a change in humidity under an environment of low humidity or high humidity or under an environment in which there are changes in humidity is desired.

SUMMARY OF THE INVENTION

One aspect of the invention is to provide a friction member on the sliding portion of at least one of a vibration member and a contacting member which are the constituent members of a vibration wave driving apparatus, and smoothly working the sliding surface of the friction member by the pressing of a mold, thereby

obtaining the sliding surface which can be worked more easily and in a shorter time than by cutting in the prior art and which is free of irregularity and assumes proper flatness and proper surface roughness.

5 One aspect of the invention is to provide a friction member on the sliding portion of at least one of a vibration member and a contacting member which are the constituent members of a vibration wave driving apparatus, the friction member being formed of a resin composition containing heat resisting resin and carbon
10 fiber of the mesophase pitch origin, and stable driving can be realized even under an environment of low humidity or high humidity.

15 BRIEF DESCRIPTION OF THE DRAWINGS

 Fig. 1A is a cross-sectional view showing the construction of a vibration type motor which is a first embodiment of the present invention, and Fig. 1B is an enlarged view of the sliding portion of the motor.

20 Fig. 2 is an enlarged view of the sliding portion of a vibration type motor which is a fourth embodiment of the present invention.

 Fig. 3 is an enlarged view of the sliding portion of a vibration type motor which is a fifth embodiment
25 of the present invention.

 Figs. 4A and 4B are illustrations showing a method of working the friction member of the motor of the

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fourth embodiment.

Figs. 5A and 5B are illustrations showing a method of working the friction member of the motor of the fifth embodiment.

5 Fig. 6 is a perspective view of a vibration type motor according to the prior art.

Fig. 7 is an enlarged view of the sliding portion of the vibration type motor according to the prior art.

10 Fig. 8 is a cross-sectional view showing another embodiment of the vibration wave motor of the present invention.

15 Fig. 9 is a graph showing the relation between the abrasion loss and humidity of a frictional material consisting of fluororesin containing carbon fiber according to a sixth embodiment of the present invention.

20 Fig. 10 is a graph showing the relation between the abrasion loss and humidity of a frictional material consisting of polyimide resin containing carbon fiber according to a seventh embodiment of the present invention.

25 Fig. 11 is a schematic view of an instrument having the vibration wave motor shown in Fig. 8 as a drive source.

DESCRIPTION OF THE PREFERRED EMBODIMENTS

[First Embodiment]

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Fig. 1A shows a ring type vibration type motor (vibration type driving apparatus) which is a first embodiment of the present invention. In Fig. 1A, the reference numeral 3 designates a ring-like elastic member formed of a metal or the like. A group of piezoelectric elements 4 as electro-mechanical energy conversion elements comprising two groups of piezoelectric elements alternately polarized and disposed in a ring-like shape are concentrically adhesively secured to the underside of the elastic member 3. Also, the upper portion of the elastic member 3 is formed into a comb-tooth-like shape to enlarge vibration amplitude, and a friction member 6 is concentrically adhesively secured to the upper surface of this comb-tooth-like portion. The elastic member 3, the group of piezoelectric elements 4 and the friction member 6 together constitute a vibration member 1. Also, the inner peripheral portion of the elastic member 3 is attached to a base member 10 by a screw 13, and the entire vibration member 1 is fixed to the base member 10.

The reference numeral 14 denotes a moving member (contacting member). This moving member 14 has a ring portion 2 and a disc-like spring portion 5 having its inner peripheral boss portion 7 spline-coupled to a motor output shaft 11, and the ring portion 2 is brought into pressure contact with the vibration member

bearing 9 of the motor output shaft 11 and produces the above-mentioned pressing force.

10 elements 4, a circumferential travelling wave is
created in the elastic member 3 and the friction member
6. Therefore, the moving member 14 which is in
pressure contact with the friction member 6 is
rotatively driven while sliding relative to the
15 friction member 6 by the friction thereof with this
friction member 6, and this rotation is transmitted to
the motor output shaft 11 through the spring portion 5
and the boss portion 7 and is taken out as a motor
output, and is used as a drive force for an object to
20 be driven in an apparatus (for example, the
photosensitive drum of a copying apparatus).

order of 3 μm . Also, the surface of contact with the friction member 6 on the moving member 14 (ring portion 2) is worked to the range of flatness of 10 μm or less

and surface roughness R_{max} of 0.2 to 2 μm suitable for motor driving by polishing.

On the other hand, as the friction member 6, use is made of 80 % by weight of PTFE resin and 20 % by weight of carbon fiber uniformly dispersed by a mixer, and press-formed, and thereafter heat-treated at 380°C.

However, as the materials of the moving member and the friction member in the present invention, use may be made of other materials than the materials in the present embodiment, for example, a metallic material or a ceramic material as the material of the moving member, and other resin than fluororesin as the base material of the friction member 6.

Fig. 1B shows the sliding portion of the motor of the present embodiment on an enlarged scale. The friction member 6 is in pressure contact with the moving member 14 (ring portion 2) on the surface (sliding surface) of the sliding portion (contact portion) 15 thereof.

In the present embodiment, the surface of the sliding portion 15 is pressed by a method of pressing a mold. Specifically, the flat portion of a punch which is a metallic mirror surface is pressed against the entire surface of the sliding portion 15 of the friction member 6 by pressure of the order of $4 \times 10^7 \text{ N/m}^2$, and is held for 60 seconds and is smoothed.

Thereby, the surface of the sliding portion 15 is worked into a smooth surface which is a surface suitable for motor driving and in which carbon fiber and PTFE resin are constructed substantially on the same surface and which is free of waviness, specifically a surface having flatness of 10 μm or less and surface roughness R_{max} within a range of 5 μm or less.

Accordingly, it becomes possible to efficiently transmit minute displacement created on the surface of the sliding portion 15 by the group of piezoelectric elements 4 to the moving member 14 and moreover, it is possible to work workpieces more easily and within a shorter time than in the working by grinding or polishing and without any irregularity between the workpieces. Further, it is possible to obtain the above-described smooth surface to thereby obtain the stable output of the vibration type motor which has been impossible by the cutting work alone.

[Second Embodiment]

While the first embodiment has been described with respect to a case where a mold is simply pressed to thereby work the surface of the sliding portion 15 of the friction member 6, the surface of the sliding portion 15 may be worked by an ultrasonic working method. The construction of the vibration type motor and the shape of each constituent member in this

embodiment are similar to those in the first embodiment.

Specifically, the flat portion of an ultrasonic working horn which is a metallic mirror surface member
5 is pressed against the entire surface of the sliding portion 15 with pressure of the order of 3×10^7 N/m² and ultrasonic vibration is applied thereto, and the flat portion is held for 10 seconds in a state in which the temperature of the sliding portion 15 is of the
10 order of 200 to 300°C, and is subjected to smoothing work.

Thereby, the surface of the sliding portion 15 is worked into a smooth surface which is a surface suitable for motor driving and in which carbon fiber
15 and PTFE resin are constructed substantially on the same surface and which is free of waviness, specifically a surface having flatness of 5 μm or less and surface roughness Rmax within a range of 3 μm or less.

20 Accordingly, it becomes possible to more efficiently transmit minute displacement created on the surface of the sliding portion 15 by the group of piezoelectric elements 4 to the moving member 14 than in the case of the first embodiment and moreover, it is
25 possible to work workpieces more easily and within a shorter time than in the working by grinding or polishing and without any irregularity between the

[Third Embodiment]

10 The construction of the vibration type motor and the
shape of each constituent member in this embodiment are
similar to those in the first embodiment.

20 Thereby, as in the second embodiment, the surface of the sliding portion 15 is worked into a smooth surface having flatness of 5 μm or less and surface roughness R_{max} within a range of 3 μm or less.

Accordingly, it becomes possible to more
25 efficiently transmit minute displacement created on the
surface of the sliding portion by the group of
piezoelectric elements 4 to the moving member 14 than

in the case of the first embodiment and moreover, it is possible to work workpieces more easily and within a shorter time than in the working by grinding or polishing and without any irregularity between the workpieces. Further, it is possible to obtain the stable output of the vibration type motor which has been impossible by the cutting work alone.

[Fourth Embodiment]

Fig. 2 shows the sliding portion of a vibration type motor which is a fourth embodiment of the present invention on an enlarged scale. The construction of the vibration type motor and the shape of each constituent member in this embodiment are basically similar to those in the first embodiment and therefore, common constituents are given the same reference characters as those in the first embodiment and need not be described.

In the present embodiment, the moving member 14 and a friction member 6' are made of the same materials as in the first embodiment, but the sliding portion 16 of the friction member 6' is formed into a bank type having ring-like grooves on the outer peripheral side and inner peripheral side thereof, and the surface (sliding surface) of this bank type sliding portion 16 is in pressure contact with the moving member 14 (ring portion 2).

The inner diameter side ridgeline 16a of the bank

type sliding portion 16 is located more diametrically outside than the inner diameter side ridgeline 2a of the ring portion 2, and the outer diameter side ridgeline 16b of the bank type sliding portion 16 is located more diametrically inside than the outer diameter side ridgeline 2b of the ring portion 2. Therefore, the surface of the bank type sliding portion 16 is in non-contact with the ridgelines 2a and 2b of the ring portion 2.

Thereby, the hard and sharp ridgelines 2a and 2b of the ring portion 2 made of aluminum containing silicon can be prevented from bearing against the bank type sliding portion 16 at a certain angle even if the ring portion 2 is deformed by vibration. Accordingly, it is possible to prevent the concentration of stress in the bank type sliding portion 16 by the bearing against the ridgelines 2a and 2b of the ring portion 2, sudden abrasion caused thereby, and further a reduction in the starting torque and the instability of the output of the motor due to the unstable contact between the ridgelines 2a, 2b and the bank type sliding portion 16.

Also, in the present embodiment, the bank type sliding portion 16 is formed by an ultrasonic working method. Specifically, as shown in Fig. 4A, a bank type ultrasonic working horn 19 made of a metal is pressed against the upper surface of the friction member 6'

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it is possible to prevent the concentration of stress in the bank type sliding portion 18 by the bearing against the ridgelines 2a and 2b' of the ring portion 2 and sudden abrasion caused thereby and further, a
5 reduction in the starting torque and the instability of the output of the motor due to the unstable contact between the ridgelines 2a, 2b' and the bank type sliding portion 18.

Also, in the present embodiment, the bank type
10 sliding portion 18 is formed by an ultrasonic working method. Specifically, as shown in Fig. 5A, a bank type ultrasonic working horn 22 made of a metal is first pressed against the surface of the friction member 6" with pressure of the order of 5×10^7 N/m² and
15 ultrasonic vibration is applied thereto, whereby the friction member 6" is held for 10 to 30 seconds in a state in which the temperature of the friction member 6" is 200 to 300°C, thereby forming a bank type sliding portion 18 of a height of 50 to 200 μm. Subsequently,
20 as shown in Fig. 5B, a flat type ultrasonic working horn 23 which is a mirror surface member made of a metal is pressed against the surface 21 of the bank type sliding portion 18 with pressure of the order of 3×10^7 N/m², and ultrasonic vibration is applied thereto,
25 whereby the bank type sliding portion 18 is held for 10 seconds in a state in which the temperature of the bank type sliding portion 18 is 200 to 300°C, thereby

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effecting the smoothing of the surface 21.

Thereby, the surface 21 of the bank type sliding portion 18, like the sliding portion 15 described in the second embodiment, is worked into a smooth surface which is suitable for motor driving and in which carbon fiber and PTFE resin are constructed substantially on the same surface and which is free of waviness.

Accordingly, it becomes possible to efficiently transmit minute displacement created on the surface 21 of the bank type sliding portion 18 by the group of piezoelectric elements 4 to the moving member 14 and moreover, it is possible to work workpieces more easily and within a shorter time than in the working by grinding or polishing and without any irregularity between the workpieces. Further, it is possible to obtain the stable output of the vibration type motor which has been impossible by the cutting work alone.

As a method of working what provides the same shape, flatness and surface roughness as those of the bank type sliding portion 18 in the present embodiment, it is possible to use a press molding method. Also, in a method of pressing a mold, it is possible to give an appropriate condition such as heating to thereby work substantially the same shape as that of the bank type sliding portion 18.

Also, each of the above embodiments has been described with respect to a case where the vibration

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more easily and within a shorter time than when the mold is simply pressed.

Also, if the sliding surface of the friction member is worked by a press molding method or an ultrasonic working method, a sliding surface free of irregularity can be obtained more easily and within a shorter time than when the mold is simply pressed in a high atmosphere.

Further, if the sliding surface of the friction member is formed into a shape which is in non-contact with at least one ridgeline portion on the sliding portion of a partner member to be contacted (for example, a contacting member made of aluminum or the like), the friction member can be prevented from being shaved by the hard and sharp ridgeline portion of the partner member, or a reduction in the starting torque and the instability of the output of the driving apparatus by unstable contact can be prevented.

Other example of the friction member will now be described.

Another friction member for the vibration wave motor is characterized by the use of a material comprising a resin composition containing heat-resisting resin and mesophased pitch carbon fiber.

That is, this friction member is characterized in that the kind of the carbon fiber contained in the material thereof is a resin material limited to

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often distinguished by the name of the raw material.

At present, commercially available on the industrial scale are two kinds, i.e., PAN carbon fiber using acryl fiber (polyacrylonitrile, hereinafter referred to as "PAN") as the starting raw material, and pitch carbon fiber made from a pitch obtained by polymerizing tar obtained during the dry distillation of coal or the residual oil of the distillation or heat decomposition of crude oil by heat treatment, and PAN carbon fiber overwhelmingly occupies the share, but pitch carbon fiber is commercially most available as inexpensive and universal carbon fiber. It is further classified into what was subjected to graphitic treatment and became graphitic fiber, and carbonic fiber which is not subjected to graphitic treatment.

Pitch carbon fiber is further divided into two, and carbon fiber made from mesophased pitch producing liquid crystal phase (mesophase) optically exhibiting anisotropy when pitch is heated and changes from the liquid phase to the solid phase is called mesophased pitch carbon fiber, and what does not produce mesophase and is made from optically isotropic pitch is called isotropic pitch carbon fiber, and these two are distinguished from each other. This difference between anisotropy and isotropy is remarkable in the orientation of the minute texture (crystal) of the carbon material and the degree thereof, and greatly

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is provided on the frictional contacting portion of at least one of the vibration member and the contacting member.

Fig. 8 is a cross-sectional view showing an embodiment of the vibration wave motor of the present invention. In Fig. 8, the reference numeral 101 designates a vibration member comprising piezoelectric elements 104 as two groups of ring-like electro-mechanical energy conversion elements polarized into a plural portion and secured to one end surface of a ring-like metallic elastic member 103 formed of stainless steel by a heat-resisting epoxy resin adhesive agent, and a friction member 105 likewise secured to the other end surface of the metallic elastic member 103.

On the other hand, a friction member 106a is provided on the frictional sliding surface of a ring-like contacting member 106 formed of an aluminum alloy on a moving member 102 side. The contacting member 106 is mounted on a support member 108 through a rubber ring 107, and the support member 108 is fixed to an output shaft 112 by a screw 111. The friction member 105 of the vibration member 101 and the friction member 106a of the contacting member 106 are in contact with each other to thereby form a frictional sliding surface, and are axially pressed by a pressing leaf spring 116 with a load of 5 kgf in total. The

reference numeral 109 denotes a bearing, the reference numeral 113 designates a disk plate for fixing the vibration member 101, reference numeral 118 designates a cover, the reference numerals 114 and 115 denote pressure-giving collars, and the reference numeral 117 designates a collar fixed to the output shaft 112 by a screw 111a.

Showing an example of the shape of the friction member 105, a circumferential level difference 105a is provided on the friction member 105, and the height C thereof is 0.15 mm. Also, the width a of the portion of contact (sliding surface) between the friction members 105 and 106a of Fig. 8 is 0.8 mm, and the diameter b of the portion of contact (sliding surface) is 30 mm.

When an AC voltage of a frequency inherent to the vibration member 101 is applied to the two groups of piezoelectric elements 104 alternately polarized in the direction of thickness thereof, the vibration member 101 causes resonance and a travelling vibration wave is created in the circumferential direction thereof, and a frictional force acts on the friction member 106a through the friction member 105, and the moving member 102 is rotatively driven.

The friction member may be used as both or one of the friction members 105 and 106a. When the friction member of the present invention is used as one of them, an ordinary friction member can be used as the other.

manner.

Carbon fiber (short fiber) shown in Table 1 and 20
% by weight of carbon fiber (short fiber) subjected to
graphitic treatment were uniformly dispersed in and
5 mixed with 80 % by weight of polytetrafluoroethylene
(hereinafter referred to as PTFE) resin powder,
whereafter they were pressed with pressure of 500 kg/cm²
to thereby make a molded body having a diameter of 8
cm, an inner diameter of 1 cm and a height of 10 cm,
10 and it was baked at a temperature of 360°C for 3 hours
to thereby obtain a cylindrical baked member. The
cylindrical baked member was cut by a cutting apparatus
to thereby make a cylindrical sheet having a thickness
of 0.5 mm. The sheet was punched and was used as the
15 ring-like friction member 105 of Fig. 8.

The dimensions of the short fiber used were a diameter of about 10 μm to 13 μm and a length of about 100 μm to 130 μm .

Table 1

Sample No.	kind of carbon fiber
1	mesophased pitch (carbonic)
2	mesophased pitch + graphitic treatment (graphitic)
3*	isotropic pitch (carbonic)
4*	isotropic pitch + graphitic treatment (graphitic)
5*	polyacrylonitrile (carbonic)

(Note 1) In Table 1, Nos. 1 and 2 show embodiments and Nos. 3, 4 and 5 (mark *) show comparative examples.

(Note 2) In Table 1, the mesophased pitch carbon fiber of sample No. 1 is trade name MC-249 produced by Osaka Gas Co., Ltd.

The mesophased pitch + graphitic treatment of sample No. 2 is trade name MG II-249 produced by Osaka Gas Co., Ltd.

The isotropic pitch carbon fiber of sample No. 3 is trade name SG-249 produced by Osaka Gas Co., Ltd.

The isotropic pitch + graphitic treatment of sample No. 4 is trade name LXX-941 produced by Osaka Gas Co., Ltd.

The polyacrylonitrile carbon fiber of sample No. 5

is trade name Toreka MLD-300 produced by Toray Industries, Inc.

In Table 1, Nos. 1, 3 and 5 are fiber which was carbonized and became carbonic after each raw material was insolubilized or stabilizing-treated (treated at 800 to 1200°C in inactivated ambient gas) after fiber forming, and Nos. 2 and 4 are fiber which was further subjected to graphitic treatment at 2000 to 3000°C in inactivated ambient gas and became carbonic.

10 The shape of the friction member 105, as shown in
Fig. 8, is such that the friction member 105 is
provided with a circumferential level difference 105a,
the height of which is 0.15 mm. Also, the width a of
the portion of contact (sliding surface) between the
15 friction members 105 and 106a is 0.8 mm and the
diameter b of the portion of contact (sliding surface)
is 30 mm.

The friction member 105 was fixed to a metallic elastic member 103 formed of stainless steel by an adhesive agent (of the epoxy thermosetting type), and as the friction member 106a of the contacting member 106 formed of aluminum, use was made of one formed of tungsten carbide containing cobalt by thermal spraying to make the vibration wave motor shown in Fig. 8.

25 The evaluation of the friction member described
below is obtained as a value obtained by the vibration
wave motor being driven at 300 rpm in rotating speed

and torque of 300 g-cm for 100 hours on end. The result is shown in Fig. 9.

Fig. 9 shows the abrasion loss of the friction member 105 formed of PTFE material containing the carbon fiber of Table 1 after 100 hours of driving when the humidity was changed to 1 % to 95 % at a temperature of 40°C. The abrasion loss is a value obtained by measuring the height of the level difference 105a of Fig. 8 by a height gauge, and subtracting the height after driving in advance from the height of the level difference 105a before driving.

As can be seen from Fig. 9, the friction members of Nos. 1 and 2, as compared with the friction members of Nos. 3 to 5, are very small in the change (increase) in abrasion at both low humidity and high humidity, as compared with humidity of 40 to 70 %.

Also, No. 2, as compared with No. 1, is generally great in abrasion loss. No. 2 is graphitic and relatively soft and its abrasion loss seems to have become relatively great.

[Seventh Embodiment]

As in Embodiment 6, by the use of the carbon fibers of Nos. 1 to 5 shown in Table 1, the resin material was made into polyimide resin, and as in Embodiment 6, 88 % by weight of polyimide resin powder and 12 % by weight of carbon material were uniformly dispersed and mixed together and a cylindrical baked

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by a photocoupler 126, and the rotation and stoppage of the motor are controlled, for example, for auto-focusing.

As described above, according to the sixth and seventh embodiments, mesophased pitch carbon fiber is used as the carbon fiber contained in the resin which is the friction member, whereby the long-term use of the vibration wave motor under an environment of low humidity or high humidity becomes possible and in addition, the stabilization of the life of the vibration wave motor under an environment having a humidity change also becomes possible, and the reliability of the vibration wave motor is more enhanced.

15 Also, there can be provided an apparatus using a
vibration wave motor excellent in the friction
characteristic to the above-mentioned humidity change.